

## Reduced stress method for Class 4 steel section

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## Summary

Design of steel cross sections with thin plates has to take in account the effect of local instability that reduce the ultimate resistance.

Eurocode 3 classifies these cross-sections as Class 4 cross-sections and in part 1-5 it shows two procedures to take in account local buckling effects in the ULS resistance evaluation:

- Effective cross section method
- Reduced stress method

Scope of this paper is to show the application of the Reduced stress method to a real case (main roof of High Velocity railway station in Florence, Italy) and some considerations about the relationship with Effective cross section's method.

**Keywords:** Effective cross section method, Reduced stress method, Eurocode 3, Class 4 steel cross-section.

## 1. Introduction

Eurocode [1], defines 4 classes of cross-sections according their capacity to develop plastic moment resistance. For class 4 cross-section design resistance  $R_d$  is limited by local buckling resistance and it is lower than calculated one adopting full-plastic or elastic distribution stresses.

To determine correctly design resistance  $R_d$  of class 4 cross-section Eurocode shows 2 methods [2]:

- 1) Effective cross section's method;
- 2) Reduced stress method.

These methods are analyzed and compared with observations on their application field and their correlations. It is possible find that in some cases Reduced stress method are more conservative than effective cross-section one.

Finally it is briefly showed an application of Reduced stress method in a real case: the design of High Velocity Railway Station roof in Florence (Italy) [3].

## 2. Method description, observations and comparison

Eurocode [1] defines 4 classes of cross-sections according their capacity to develop plastic moment resistance (this capacity is limited by local buckling phenomenon).

According this classification Class 4 cross-sections are those in which local buckling occur before the attainment of yield stress in one or more parts of cross-section.

The classification of a cross-section depends on:

- a) the width to thickness ratio of the parts subjected to compression;
- b) the tipology of component plates of cross-section (internal or outstand compression plates);



- c) the distribution of direct stresses  $\sigma$  on each plate;
- d) the mechanical characteristics of steel.

On the basis of these informations it be possible classify each part of cross-section. Cross-section class will be the highest class of its compression parts.

It is important to observe that the classification depends only on the direct stresses  $\sigma_x$ . Shear stresses  $\tau$  and stresses acting parallel to cross-section plane ( $\sigma_z$ ) are not considered.

Design resistance  $R_d$  of class 4 cross-section is limited by local buckling resistance and it is lower than calculated one adopting full-plastic or elastic distribution stresses.

To determine correctly design resistance  $R_d$  of class 4 cross-section Eurocode shows 2 methods [2]:

- 3) Effective cross section's method;
- 4) Reduced stress method.

In effective cross section's method the portions of plates that are subject to local buckling is removed from cross-section to obtain a residual cross-section named "effective" cross-section. Design resistance is determined from this effective cross-sections assuming Class 3 for it.

Some observations:

- 1) As in cross-section classification procedure, effective cross section's method depends on:
  - a) the width to thickness ratio of the parts subjected to compression;
  - b) the tipology of component plates of cross-section (internal or outstand compression plates);
  - c) the distribution of direct stresses  $\sigma$  on each plate;
  - d) the mechanical characteristics of steel.
- 2) The reduction from gross cross-section to effective cross-section depends only to direct stresses  $\sigma_x$ . Shear stresses  $\tau$  and stresses acting parallel to cross-section plane ( $\sigma_z$ ) and their influence are treated separately.
- 3) Stress distribution necessary to define  $\psi$  parameter would have to be based on:
  - a) gross cross-section for stress distribution on flange plates;
  - b) section with effective flange plates for stress distribution on web plates.
- 4) The reduction factor does not depend on the real intensity of  $\sigma_x$  stresses but only from its distribution. Eurocode allows to take in account stresses intensity for the classification of cross-section (and not for instability resistance evaluation) (see [1] p. 5.5.2(9)) and for effective cross-section evaluation (see [2] p. 4.4(4)). In this way the procedure becomes iterative and the heavier computational effort finds justification only for elements subjected to low stresses.
- 5) Effective cross-section derived from symmetrical gross cross-section can be without symmetry so in verifications it is necessary take in account an eccentricity  $e_N$  (distance between center of mass of gross and effective cross-sections) of axial force and its derived supplementary bending moment  $\Delta M = N e_N$ .
- 6) Effective cross section's method can be used only when (see [2] p. 2.3(1)):
  - a) cross-section plates are rectangular;
  - b) flange plates are parallel;
  - c) any unstiffened open holes are little.

Alternative method is named as Reduced stress method (see [2] p.10).

The method:

- 1) allows to take in account of direct stresses  $\sigma_x$ , shear stresses  $\tau$ , stresses  $\sigma_z$  acting parallel to cross-section plane;
- 2) allows to define the acceptability of cross-section stresses distribution from the combined point of view of resistance and instability by means of the acceptability of stresses distribution of single cross-section plates;



- 3) allows to adopt as reference the stresses distribution derived from gross cross-section without iterative procedure and without additional eccentricity  $e_N$ ;
- 4) is the generalization of the previous effective cross-sections method.

## 2.1 Method analytical description

Resistance verification requires that

$$\left( \frac{F_{Ed}}{F_{Rd}} \right)_{sect} \leq 1 \quad (1)$$

and assuming  $\left( \frac{F_{Ed}}{F_{Rd}} \right)_{sect} = \max_i \left( \frac{F_{Ed}}{F_{Rd}} \right)_i$  it is necessary to check for each  $i$ -th plate of cross-section that

$$\left( \frac{F_{Ed}}{F_{Rd}} \right)_i \leq 1 \quad \forall i \quad (2)$$

Naming with  $\alpha_{ult,k}$  the minimum load amplifier for the design loads to reach the characteristic value of resistance of the most critical point of the plate, design resistance can be described as

$$F_{Rd} = \frac{\alpha_{ult,k} F_{Ed}}{\gamma_M} \quad (3)$$

Now it is possible take in account plate buckling adopting a reduction factor  $\rho$  which depends on plate slenderness  $\lambda$ .

$$F_{Rd} = \frac{\rho \alpha_{ult,k} F_{Ed}}{\gamma_M} \quad (4)$$

The acceptability criteria (2) can be formulated as

$$\frac{\rho \alpha_{ult,k}}{\gamma_M} \geq 1 \quad (\text{for each plates of cross-section}) \quad (5)$$

The minimum load amplifier for the design loads to reach the characteristic value of resistance (without buckling effects)  $\alpha_{ult,k}$  is evaluated from stresses field by applying VonMises criteria:

$$\frac{1}{\alpha_{ult,k}^2} = \left( \frac{\sigma_{x,Ed}}{f_{yk}} \right)^2 + \left( \frac{\sigma_{z,Ed}}{f_{yk}} \right)^2 - \left( \frac{\sigma_{x,Ed}}{f_{yk}} \right) \left( \frac{\sigma_{z,Ed}}{f_{yk}} \right) + 3 \left( \frac{\tau_{Ed}}{f_{yk}} \right)^2 \quad (6)$$

Reduction factor  $\rho$  is evaluated following these steps:

- determination of the elastic critical buckling stress for each single stress field:

$$\sigma_{cr,x} \quad \sigma_{cr,z} \quad \tau_{cr}$$

- determination of the minimum load amplifier for the design loads to reach the elastic critical load of the plate under the single stress field:

$$\alpha_{cr,x} = \sigma_{cr,x} / \sigma_{x,Ed} \quad \alpha_{cr,z} = \sigma_{cr,z} / \sigma_{z,Ed} \quad \alpha_{cr,\tau} = \tau_{cr} / \tau_y$$

- determination of load amplifier for the design loads to reach the elastic critical load of the plate under the complete stress field:

$$\frac{1}{\alpha_{cr}} = \frac{1+\psi_x}{4\alpha_{cr,x}} + \frac{1+\psi_z}{4\alpha_{cr,z}} + \left[ \left( \frac{1+\psi_x}{4\alpha_{cr,x}} + \frac{1+\psi_z}{4\alpha_{cr,z}} \right)^2 + \frac{1-\psi_x}{2\alpha_{cr,x}^2} + \frac{1-\psi_z}{2\alpha_{cr,z}^2} + \frac{1}{\alpha_{cr,\tau}^2} \right]^{1/2} \quad (7)$$

Where  $\psi_x$  e  $\psi_z$  coefficients take in account the stresses distribution on the plate (with maximum value  $\sigma_{x,Ed}$  e  $\sigma_{z,Ed}$  respectively, it is assumed that shear stress  $\tau$  id uniform).

- determination of plate slenderness  $\lambda$  :



$$\bar{\lambda}_p = \sqrt{\frac{\alpha_{ult,k}}{\alpha_{cr}}} \quad (8)$$

- determination of buckling reduction factors for each stress field and for the complete stress field:

$$\rho_x = \rho_x(\bar{\lambda}_p) \quad \rho_z = \rho_z(\bar{\lambda}_p) \quad \chi_w = \chi_w(\bar{\lambda}_p) \quad (9)$$

- determination of buckling reduction factors for the complete stress field:

$$\rho = \min(\rho_x, \rho_z, \chi_w) \quad (10)$$

## 2.2 Elementary cases

Analyzing elementary cases it can be show that from Reduced stress method it is possible to obtain the Effective cross-section method.

### 2.2.1 Case 1: Plated subjected to compression direct stresses

Plate (b x t) subjected to compression direct stresses field ( $\sigma_x$ ) only:

$$\begin{aligned} \sigma_{xE} \neq 0 & \quad (= N_{sd} / A = N_{sd} / (bt)) & \quad \sigma_{zE} = \tau_{Ed} = 0 & \quad \alpha_{ult,k} = f_{yk} / \sigma_{xE} \\ \alpha_{cr,x} = \sigma_{cr,x} / \sigma_{xE} & & \quad \alpha_{cr,z} = \sigma_{cr,z} / \sigma_{zE} \rightarrow \infty & \quad \alpha_{cr,\tau} = \tau_{cr} / \tau_{Ed} \rightarrow \infty \\ \sigma_{cr,x} = k_\sigma(\psi) \sigma_E & \quad \text{elastic critical plate buckling stress} & & \\ \text{where } \sigma_E = \frac{\pi^2 Et^2}{12(1-\nu^2)b^2} & \quad \text{Eulerian elastic critical plate buckling stress} & & \end{aligned}$$

$k_\sigma(\psi)$  = buckling factor (according to stresses distribution)

from (7)

$$\alpha_{cr} = \alpha_{cr,x} = k_\sigma(\psi) \frac{\sigma_E}{\sigma_{xE}} = k_\sigma(\psi) \frac{\sigma_E}{\frac{f_{yk}}{\alpha_{ult,k}}} = k_\sigma(\psi) \frac{\sigma_E}{\frac{235}{\varepsilon^2 \alpha_{ult,k}}} \quad \bar{\lambda}_p^2 = \frac{\alpha_{ult,k}}{\alpha_{cr}} = \frac{235}{k_\sigma(\psi) \sigma_E \varepsilon^2}$$

Obtaining the expression of effective cross-section method (see [2] p. 4.4(1))

$$\begin{aligned} \bar{\lambda}_p &= \frac{\frac{b}{t}}{28,4 \varepsilon \sqrt{k_\sigma(\psi)}} \quad \text{that it is used to calculate } \rho_x (= \rho) \\ \frac{\rho \alpha_{ult,k}}{\gamma_M} \geq 1 & \quad \rightarrow \quad \frac{\rho f_{yk}}{\sigma_{xE} \gamma_M} \geq 1 \quad \rightarrow \quad \frac{A \rho f_{yk}}{A \sigma_{xE} \gamma_M} \geq 1 \quad \rightarrow \quad \frac{A \rho f_{yk}}{\gamma_M} \geq A \sigma_{xE} \rightarrow \\ \rightarrow \quad A_{eff} f_{yd} \geq A \sigma_{xE} & \quad \rightarrow \quad N_{Rd} \geq N_{Sd} \end{aligned}$$

### 2.2.2 Case 2: Plated subjected to tension direct stress

Plate (b x t) subjected to tension direct stresses field ( $\sigma_x$ ) only:

$$\begin{aligned} \sigma_{xE} \neq 0 & \quad (= N_{sd} / A = N_{sd} / (bt)) & \quad \sigma_{zE} = \tau_{Ed} = 0 & \quad \alpha_{ult,k} = f_{yk} / \sigma_{xE} \\ \alpha_{cr,x} = \sigma_{cr,x} / \sigma_{xE} & \rightarrow \infty \quad (\text{tension plate has no buckling}) & & \quad \alpha_{cr,z} = \sigma_{cr,z} / \sigma_{zE} \rightarrow \infty \\ \alpha_{cr,\tau} = \tau_{cr} / \tau_{Ed} & \rightarrow \infty & & \\ \text{from (7)} & & & \end{aligned}$$

$$\alpha_{cr} \rightarrow \infty \quad \bar{\lambda}_p = \sqrt{\frac{\alpha_{ult,k}}{\alpha_{cr}}} = 0$$

Thus  $\rho = \rho_x = 1$  and then

$$\begin{aligned} \frac{\rho \alpha_{ult,k}}{\gamma_M} \geq 1 & \quad \rightarrow \quad \frac{f_{yk}}{\sigma_{xE} \gamma_M} \geq 1 \quad \rightarrow \quad \frac{A f_{yk}}{A \sigma_{xE} \gamma_M} \geq 1 \quad \rightarrow \quad \frac{A f_{yk}}{\gamma_M} \geq A \sigma_{xE} \rightarrow \\ \rightarrow \quad A f_{yd} \geq A \sigma_{xE} & \quad \rightarrow \quad N_{Rd} \geq N_{Sd} \end{aligned}$$



### 2.2.3 Case 3: Plated subjected to shear stresses

Plate ( $h_w \times t$ ) subjected to shear stresses field ( $\tau$ ) only:

$$\tau_{Ed} \neq 0 \quad (= V_{Ed} / A = V_{Ed} / (h_w t)) \quad \sigma_{xE} = \sigma_{zE} = 0 \quad \alpha_{ult,k} = \frac{f_{yk}}{\sqrt{3}\tau_{Ed}}$$

$$\alpha_{cr,x} = \sigma_{cr,x} / \sigma_{xE} \rightarrow \infty \quad \alpha_{cr,z} = \sigma_{cr,z} / \sigma_{zE} \rightarrow \infty \quad \alpha_{cr,\tau} = \tau_{cr} / \tau_{Ed}$$

$$\tau_{cr} = k_\tau \sigma_E \quad \text{elastic critical plate buckling shear stress}$$

$$\text{where } \sigma_E = \frac{\pi^2 E t^2}{12(1-\nu^2)h_w^2} \quad \text{Eulerian elastic critical plate buckling stress}$$

$$k_\tau(h_w/t) = \text{buckling factor (according to height to thickness ratio)}$$

from (7)

$$\alpha_{cr} = \alpha_{cr,t} = k_\tau \frac{\sigma_E}{\tau_{Ed}} = k_\tau \frac{\sigma_E}{\frac{f_{yk}}{\sqrt{3}} \frac{1}{\alpha_{ult,k}}} = k_\tau \frac{\sigma_E}{\frac{235}{\sqrt{3}\varepsilon^2 \alpha_{ult,k}}} \quad \bar{\lambda}_p^2 = \frac{\alpha_{ult,k}}{\alpha_{cr}} = \frac{235}{\sqrt{3}k_\tau \sigma_E \varepsilon^2}$$

Obtaining the expression of shear design resistance (see [2] (5.5))

$$\bar{\lambda}_w = \frac{h_w}{37,4\varepsilon\sqrt{k_\tau}} \quad \text{that it is used to calculate } \chi_w (= \rho)$$

$$\frac{\rho \alpha_{ult,k}}{\gamma_M} \geq 1 \quad \rightarrow \quad \frac{\rho f_{yk}}{\sqrt{3}\tau_{Ed}\gamma_M} \geq 1 \quad \rightarrow \quad \frac{A \rho f_{yk}}{\sqrt{3}A \tau_{Ed}\gamma_M} \geq 1 \quad \rightarrow \quad \frac{A \rho f_{yk}}{\sqrt{3}\gamma_M} \geq A \tau_{Ed} \quad \rightarrow$$

$$\rightarrow \quad V_{Rd} \geq V_{Sd}$$

### 2.3 Stress reduction method vs. Effective cross section method

Using the two methods, the verification of a single plate subjected to a direct stresses field ( $\sigma_x \neq 0$ ,  $\sigma_z = \tau = 0$ ) leads to the same results. This may be not more true when the methods are applied to a section (= set of more plates).

While Effective cross-section method reduces the geometrical cross-section and then gets the stresses field from the given strains field, Reduced stress method first gets the stresses field from the given strains field and then gets the reduction buckling factor  $\rho$ .

As example it is possible to assume a square hollow section made by 4 plates: 2 horizontal plates ( $b_f \times t_f = 500 \times 20 \text{mm}$ ) and two vertical plates ( $b_w \times t_w = 500 \times 10 \text{mm}$ ) and subjected to compression axial force. Steel class: S355 ( $f_{yk} = 355 \text{ N/mm}^2$ ,  $\varepsilon = 0.81$ ).

According the Eurocode 3 cross-section classification method, horizontal plates have class < 4 ( $b_f/t_f = 500/20 = 25 < 42\varepsilon = 34$ ) and vertical plates have classe 4 ( $b_w/t_w = 500/10 = 50 > 42\varepsilon = 34$ ) therefore the cross-section has class 4.

Using Effective cross-section method:

$$\text{Gross cross-section area: } A = 2 \times (500 \times 10 + 500 \times 20) = 30000 \text{ mm}^2$$

$$\text{Uniform stresses distribution } \rightarrow \quad \psi = 1 \rightarrow k_\sigma = 4$$

$$\text{Vertical plate slenderness: } \bar{\lambda}_p = \frac{\frac{b_w}{t_w}}{28,4\varepsilon\sqrt{k_\sigma(\psi)}} = \frac{\frac{500}{10}}{28,4 \times 0,81 \sqrt{4}} = 1,087$$

$$\text{Plate reduction factor } \rho = \frac{\bar{\lambda}_p - 0,055(3 + \psi)}{\bar{\lambda}_p^2} = \frac{1,087 - 0,055(3 + 1)}{1,087^2} = 0,734$$

$$\text{Vertical plate effective cross-section area: } (\rho b_w) t_w =$$

$$= (0,734 \times 500) \times 10 = 367 \times 10 = 3670 \text{mm}^2$$



Effective cross-section area  $A_n = 2 \times 3670 + 2 \times 500 \times 20 = 27340 \text{ mm}^2$

Cross-section design resistance axial force (named  $N_{EFF\_Rd}$  for the used method):

$$N_{EFF\_Rd} = f_{yk} A_n = f_{yk} / \gamma_{M0} A_n = 355 / 1,05 \times 27340 \text{ mm}^2 = 9243 \text{ kN}$$

Using Reduced stress method:

Assuming  $N_{Ed} = N_{EFF\_Rd} = 9243 \text{ kN}$  the design stress is

$$\sigma_{Ed} = N_{Ed} / A = 9243000 / 30000 = 308 \text{ N/mm}^2 \text{ (uniform on all plates)}$$

Following the method procedure:

	Horizontal plate	Vertical plate
Dimensions b x t [mm]	500 x 20	500 x 10
Steel $f_{yk}$ [N/mm <sup>2</sup> ]	355	355
$\gamma_M$	1,05	1,05
$\sigma_{Ed}$ [N/mm <sup>2</sup> ]	308	308
$\alpha_{ult,k} = f_{yk} / \sigma_{xEd}$	1,15	1,15
$\sigma_E = \frac{\pi^2 Et^2}{12(1-\nu^2)b^2}$ [N/mm <sup>2</sup> ]	304	76
$\psi, k_\sigma(\psi)$	1, 4	1, 4
$\sigma_{cr,x} = k_\sigma(\psi) \sigma_E$ [N/mm <sup>2</sup> ]	1216	304
$\alpha_{cr} = \alpha_{cr,x} = \sigma_{cr,x} / \sigma_{x,Ed}$	3,95	0,99
$\bar{\lambda}_p = \sqrt{\frac{\alpha_{ult,k}}{\alpha_{cr}}}$	0,540	1,08
$\rho = \rho_x(\bar{\lambda}_p)$	1,00	0,74
$\Gamma = \frac{\rho \alpha_{ult,k}}{\gamma_M}$	1,10 > 1 → verified	0,81 < 1 → not verified

Cross-section is not verified. In fact, according Reduced stress method cross-section design resistance axial force is only  $N_{RED\_Rd} = 7470 \text{ kN}$  (with uniform stress  $\sigma_{Ed} = 249 \text{ N/mm}^2$ ). Vertical plates  $\Gamma$  ratio is 1,00 and horizontal plates  $\Gamma$  ratio is only 1,36 (they are not fully employed):

	Horizontal plate	Vertical plate
Dimensions b x t [mm]	500 x 20	500 x 10
Steel $f_{yk}$ [N/mm <sup>2</sup> ]	355	355
$\gamma_M$	1,05	1,05
$\sigma_{Ed}$ [N/mm <sup>2</sup> ]	249	249
$\alpha_{ult,k} = f_{yk} / \sigma_{xEd}$	1,43	1,43
$\sigma_E = \frac{\pi^2 Et^2}{12(1-\nu^2)b^2}$ [N/mm <sup>2</sup> ]	304	76
$\psi, k_\sigma(\psi)$	1, 4	1, 4
$\sigma_{cr,x} = k_\sigma(\psi) \sigma_E$ [N/mm <sup>2</sup> ]	1216	304
$\alpha_{cr} = \alpha_{cr,x} = \sigma_{cr,x} / \sigma_{x,Ed}$	4,88	1,22
$\bar{\lambda}_p = \sqrt{\frac{\alpha_{ult,k}}{\alpha_{cr}}}$	0,541	1,08
$\rho = \rho_x(\bar{\lambda}_p)$	1,00	0,74
$\Gamma = \frac{\rho \alpha_{ult,k}}{\gamma_M}$	1,36 > 1 → verified	1,00 > 1 → verified

To have  $N_{RED\_Rd} = N_{EFF\_Rd} = 9243$  kN it is necessary to assume a stresses field with  $\sigma_{v\_Ed} = \text{cost.} = 291$  N/mm<sup>2</sup> on vertical plates and  $\sigma_{H\_Ed} = \text{cost.} = f_{yd} = 338$  N/mm<sup>2</sup> on horizontal plates but this stresses distribution has no congruence-acceptability.

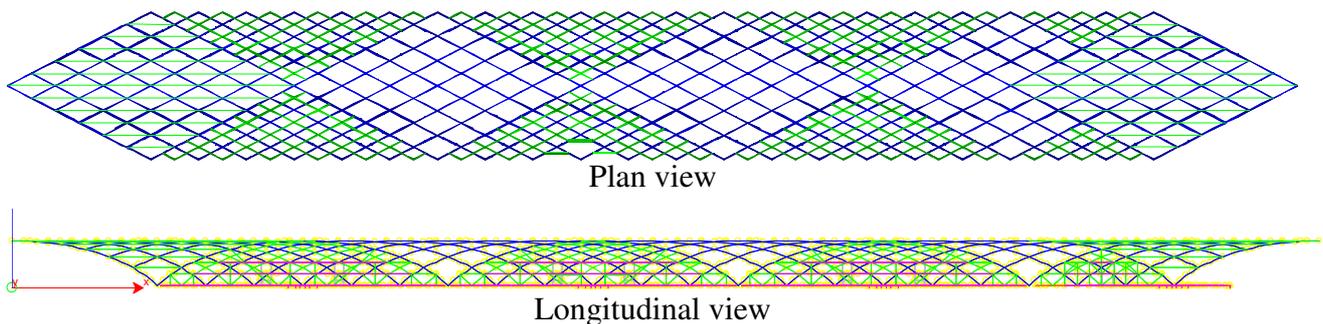
	Horizontal plate	Vertical plate
Dimensions b x t [mm]	500 x 20	500 x 10
Steel $f_{yk}$ [N/mm <sup>2</sup> ]	355	355
$\gamma_M$	1,05	1,05
$\sigma_{Ed}$ [N/mm <sup>2</sup> ]	338	249
$\alpha_{ult,k} = f_{yk} / \sigma_{xEd}$	1,05	1,43
$\sigma_E = \frac{\pi^2 E t^2}{12(1-\nu^2) b^2}$ [N/mm <sup>2</sup> ]	304	76
$\psi, k_\sigma(\psi)$	1, 4	1, 4
$\sigma_{cr,x} = k_\sigma(\psi) \sigma_E$ [N/mm <sup>2</sup> ]	1216	304
$\alpha_{cr} = \alpha_{cr,x} = \sigma_{cr,x} / \sigma_{x,Ed}$	3,60	1,22
$\bar{\lambda}_p = \sqrt{\frac{\alpha_{ult,k}}{\alpha_{cr}}}$	0,541	1,08
$\rho = \rho_x(\bar{\lambda}_p)$	1,00	0,74
$\Gamma = \frac{\rho \alpha_{ult,k}}{\gamma_M}$	1,00 > 1 → verified	1,00 > 1 → verified

This example shows that, when both methods are applicable, there are cases where Reduced stress method is more conservative than Effective cross-section method.

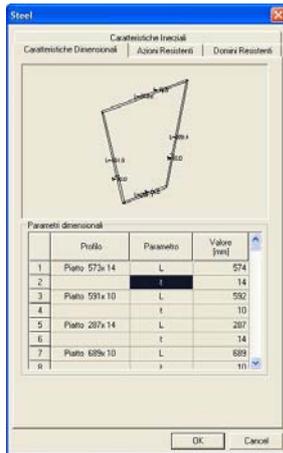
### 3. A real case: High Velocity Railway Station in Florence (Italy)

Reduced stress method has been adopted for the verification of steel member of the High Velocity Railway Roof in Florence.

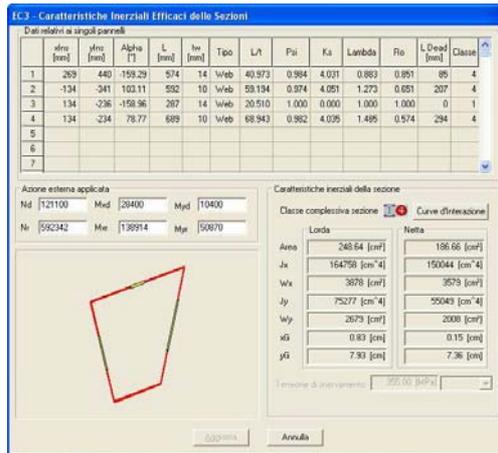
The roof structure is thought as a cylindric vault with 350m length and 52m width. At each end the roof goes on with a cantilever “nail” with 50m span. The structure is formed by a romboidal grid of plane arches. Arch vertical planes are inclined respect to roof longitudinal axe.



Arches are differentiate by primary and secondary arches. Their cross-sections have trapezoidal hollow shapes with height variable respectively from 900mm and 600mm at mid span to 1600mm and 1200mm at the supports. These box are composed by welding plate with thickness variable from 10mm to 50mm. Except to the joints at the intersection of arches, no internal ribs are planned for these plates. The 3D iperstaticity of the structure causes the presence of all six components of internal action and the use of slender plates moved the Designers to adopt the Reduced stress method for the arch verification ([3]). The constructive phase of design ([4]) has also adopted the method with the implementation of electronic sheets for the detailed analysis of check steps on single elements and of C++ routines for structural software (see following pictures).



Dimensions of cross-section plates



Cross-section classification according to EN1993

VERIFICA PANNELLO AB - EUROCODICE 3 parte 1-5 (Reduced stress method)				IC3 - Caratteristiche Inerziali Efficaci delle Sezioni			
Verifica Fucolo	0			xi1,x	1216 N/mm2	= Km*sigmaE	
gammaM	1.05 N/mm2	coefficiente di sicurezza		xi2,x	0.541	= b1/f1/28.4 f1/(k*xi1*0.5)	
fy	355 N/mm2	tensione di snervamento acciaio (R15)		xi3,x	0.905	= xi1/EC3 1-5 p.to 10(9)	
fy_fucolo	1.0 N/mm2	tensione di snervamento acciaio (R15)		xi4	1.00	= 1.0 se xi2 <= 0.673, (xi1-0.055*max(0.2*psi)/(0.9))^2 se xi2 > 0.673	
E	205000 N/mm2	modulo di Young dell'acciaio		xi_c	22.67	= alpha*(1+0.5)/(xi1-0.5)	COLUMN TYPE BUCKLING BEHAVIOUR
epsilon	0.814	modulo di Young dell'acciaio		alpha	0.21	coefficiente relativo alle imperfezioni per piatti non irrigiditi (EC3 1-5 p.to 4.3.3(5))	
v	0.3	coefficiente di Poisson		xi	259.65		
t	10 mm	spessore del pannello		xi_c	0.602	fattore di riduzione	
b	500 mm	larghezza del pannello		etf,c	0.69	ds*xi_c = (gamma_f*etf_c)^0.5	stirzo elastico critico di colonna per piatti non irrigiditi (4.5.3(2))
a	10000 mm	lunghezza del pannello		etf,b	1216	= etf_c * K*alpha*sigmaE	
(a = distanza tra due irrigidimenti trasversali)				etf	1.00	= etf_c/(etf_c-1) compresso tra 0 e 1 EC3 1.5 p.to 4.5.4	
R	38.000 m	raggio di curvatura arco		etf_c	1.0000	= (xi_c-1)/(2-1) * xi_c (EC3 1.5 p.to 4.5.4)	
la	6666687 mm4	momento d'inerzia del piatto la = 1/12 a * b^3		etf_t,x	4.88	= etf_c / alpha*Ed	= etf_c / (max(xi1,xi2))
Ncr,cr	53954 kN	azione normale critica Ncr,cr = alpha^2 E la / b^2		alpha	0.51	minimo amplificatore di carico per instabilità	
AKr,a	200000 mm2	area del piatto in direzione longitudinale AKr,a = a * t		beta	1.60	snellizzazione del piatto = (alpha*xi_c*etf_c)^1/2	EC3 1-5 p.to 10(6)
lb	333333 mm4	momento d'inerzia del piatto lb = 1/12 b * b^3		etf_t,z	-	= etf_c / alpha*Ed	
Ncr,z	7 kN	azione normale critica Ncr,z = alpha^2 E lb / a^2		alpha	0.2048	EC3 1-5 p.to 10(6)	
AKr,b	10000 mm2	area del piatto in direzione trasversale AKr,b = b * t		etf_t	4.88	minimo amplificatore di carico per instabilità	
sigma1	249.00 N/mm2	tensione nel punto 1 del pannello (+compressione, -trazione)		xi	0.51	snellizzazione del piatto = (alpha*xi_c*etf_c)^1/2	EC3 1-5 p.to 10(6)
sigma2	249.00 N/mm2	tensione nel punto 2 del pannello (+compressione, -trazione)		etf_t	0.51		
sigma3	61.43 N/mm2	tensione massima lungo z esterno 1 del pannello		etf_t	0.51		
sigma4	61.43 N/mm2	tensione massima lungo z esterno 2 del pannello		etf_t	0.51		
sigma5	61.43 N/mm2	tensione massima tangenziale totale		etf_t	0.51		
sigma6	304 N/mm2	tensione critica lungo x		etf_t	0.51		
sigma7	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma8	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma9	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma10	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma11	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma12	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma13	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma14	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma15	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma16	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma17	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma18	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma19	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma20	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma21	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma22	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma23	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma24	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma25	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma26	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma27	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma28	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma29	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma30	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma31	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma32	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma33	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma34	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma35	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma36	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma37	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma38	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma39	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma40	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma41	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma42	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma43	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma44	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma45	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma46	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma47	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma48	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma49	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma50	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma51	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma52	270 N/mm2	tensione critica lungo z		etf_t	0.51		
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sigma60	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma61	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma62	270 N/mm2	tensione critica lungo z		etf_t	0.51		
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sigma139	270 N/mm2	tensione critica lungo z		etf_t	0.51		
sigma140							